

## **AMENDMENTS TO THE SPECIFICATION**

*Please add the following paragraph:*

On page 1, immediately before "Field of the Invention", please insert the following:

### **CROSS-REFERENCE TO RELATED APPLICATION**

This is a continuation of an application filed January 2, 2002 and assigned U.S. Serial No. 10/162,535.

*Please replace Paragraphs [0029], [0030] and [0034] with the following paragraphs:*

**[0029]** The second portion 26 of the carrier 16 includes an outer flange 100 extending from first and second walls 56, 58 around the perimeter of first portion 24. Likewise, base 42 of first portion 24 extends to second portion 26 and is bordered by flange 100. Slot 102 is integrated into base 42. The slot 102 forms an opening through the carrier 16. Slot 102 is defined by a first and second pair of lateral walls 104, 106 extending from opposing edges of the slot 102. Lateral wall sections 104, 106 define a guide path 110. Slot 102 further includes opposing channels 114, 116 formed in base 42 of carrier 16. The channels 114, 116 are formed between first and second pair of lateral wall sections 104, 106. As will be described in detail, opposing channels 114, 116 are configured to accommodate gear support shaft 120. The opposite ends of guide path 110 are defined by grooves 122, 124.

**[0030]** Turning now to Figures 2 - 4, the drive assembly 30 will be further described. The drive assembly 30 is initiated by actuation of trigger 96 extending from housing 14. Motor 12 includes drive gear 88 which is splined for rotation with output gear 125. Drive assembly 30 includes gear support shaft 120 supported at opposite ends by bearings 140, 142. Gear noise and wear are reduced by supporting shaft 120 at opposite ends. Output gear 125 is disposed on support shaft 120. An offset output shaft 130 is coupled to output gear 125. The output shaft 130 is arranged to orbit support shaft 120. A follower 134 is rotatably coupled to offset output shaft 130. Follower 134 includes roller 136 and slider 138 which are received within guide path 110 of slot 102. As such, translation of follower 134 along guide path 110 encourages concurrent rotatable engagement of roller 136 and slidable engagement of slider 138 along the face of the corresponding lateral walls 104, 106. The advantage of using follower 134 including roller 136 and slider 138 is that it provides reduced gear noise and wear while providing low output shaft 130 losses.

**[0034]** Turning to Figures 6 – 8, the reciprocating saw 10 includes a primary and secondary base 144, 154. Primary base 144 includes heel portions 146 and lateral rails 148 extending from opposite edges to provide stability during operation. Secondary base 154 includes a second pair of lateral rails 156 inwardly extending from outer walls 158. Secondary base 154 is selectively

attachable to primary base 144 to provide greater surface area and stability during operation. A latch 160 is mounted to primary base 144 and provided for coupling the secondary base 154 with the primary base 144 in an engaged position. A button 162 of the latch 160 is configured to extend through an opening 166 in the housing 14 to improve accessibility. The latch 160 maintains a static position in the tool assembly by a compression spring (not shown). The spring acts along its axis between sidewall 174 of button 162 on one end and an interior wall (not shown) of primary base 144 on the opposite end which forces the latch 160 into engagement with the contact wall 172 of stop 170.